

The Piston Problem

1. Problem Statement: Determine the acceleration equation for the engine piston rod in Figure 1 as a function of crankshaft angle and crankshaft RPM. Use the acceleration equation to determine connecting rod stress at a function of RPM and crank angle (θ). Ignore connecting rod mass. Determine at the crank angle which will cause the rod to fail at 10,500 RPM.

2. Solution technique:

Using calculus, derive the following equations as a function of θ :

- a) the piston displacement equation
- b) the piston velocity equation
- c) the piston acceleration equation

Compute the force and stress equations at the minimum piston rod cross sectional area.

Compute the failure θ at RPM = 10,000.

Let θ be measured from Top Dead Center (TDC) with θ measured positively in a counter clockwise direction.

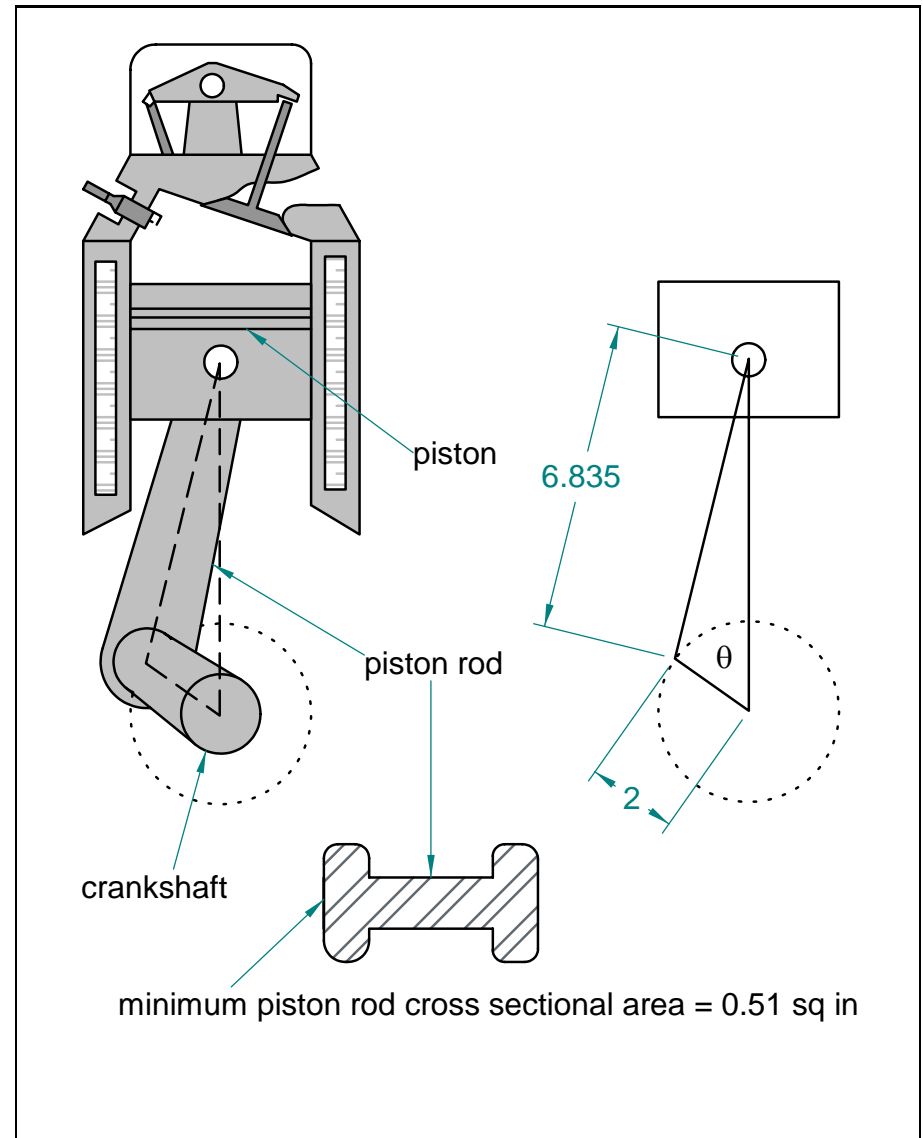
Known values:

$L := 6.835 \cdot \text{in}$ $r := 2 \cdot \text{in}$

$\text{Mass}_{\text{piston}} := 3 \cdot \text{lb}$ (with pin)

$\text{cross}_{\text{sec}} := 0.51 \cdot \text{in}^2$

Figure 1



Derive the piston displacement equation the the vertical direction as a function of crank angle (θ).

We know from Figure 2 the following relationships (where we let, for example): $\theta := 30\text{-deg}$

$$\sin(\theta) = \frac{c}{r} \quad c := \sin(\theta) \cdot r \quad \cos(\theta) = \frac{b}{r} \quad b := \cos(\theta) \cdot r$$

$$L^2 = c^2 + a^2 \quad a := (L^2 - c^2)^{\frac{1}{2}} \quad (\text{positive root only})$$

$$h := b + a \quad h = 8 \text{ in}$$

Changing the equations derived above into functions yields:

$$c(\theta) := \sin(\theta) \cdot r \quad b(\theta) := \cos(\theta) \cdot r \quad a(\theta) := (L^2 - c(\theta)^2)^{\frac{1}{2}}$$

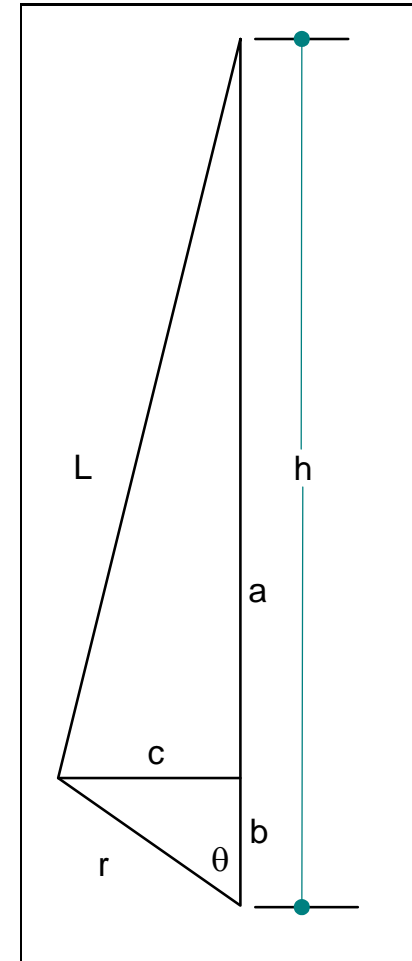
and $h(\theta) := (b(\theta) + a(\theta))$

The displacement equation (as a function of the crank angle (θ)) is therefore:

$$h(\theta) = b(\theta) + a(\theta) = \cos(\theta) \cdot r + (L^2 - c(\theta)^2)^{\frac{1}{2}} = \cos(\theta) \cdot r + (L^2 - r^2 \cdot \sin(\theta)^2)^{\frac{1}{2}}$$

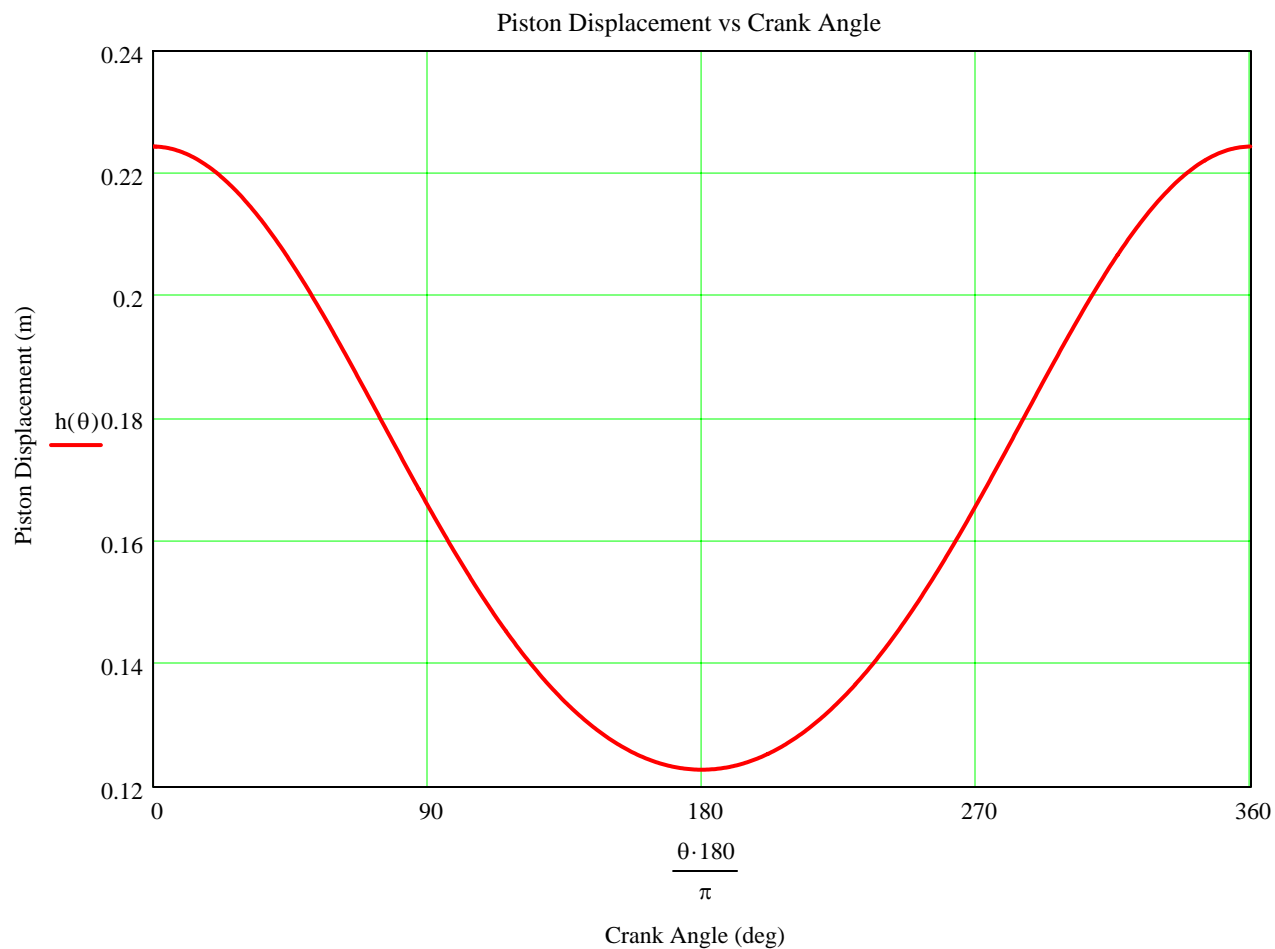
or: $h(\theta) = \cos(\theta) \cdot r + \sqrt{L^2 - r^2 \cdot \sin(\theta)^2}$

Figure 2



$$\theta := 0, .01 .. 2 \cdot \pi$$

The piston displacement (the linear distance measured from center of crankshaft to piston pin) graph relative to the crank angle in the domain of 0 to 360 degrees is shown to the right.



Calculating piston velocity by taking derivative of the displacement function with respect to time:

$$\frac{d}{dt}h(\theta) = \frac{d}{dt} \left[\cos(\theta) \cdot r + \left(L^2 - r^2 \cdot \sin(\theta)^2 \right)^{\frac{1}{2}} \right] = \frac{d}{dt}(\cos(\theta) \cdot r) + \frac{d}{dt} \sqrt{L^2 - r^2 \cdot \sin(\theta)^2}$$

breaking into 2 parts and recalling $\omega = \text{angular velocity (RPM)}$ since: $\omega = \frac{d}{dt}\theta$ (the time rate of change of the crank angle)

Part 1
$$\frac{d}{dt}(\cos(\theta) \cdot r) = \cos(\theta) \cdot \frac{d}{dt}r + r \cdot \frac{d}{dt}\cos(\theta) = \cos(\theta) \cdot 0 + r \cdot \left[- \left[\left(\frac{d}{dt}\theta \right) \cdot \sin(\theta) \right] \right] = -r \cdot \omega \cdot \sin(\theta)$$

Part 2 Remembering from calculus:
$$\frac{d}{dt}\sqrt{u} = \frac{1}{2 \cdot \sqrt{u}} \cdot \frac{d}{dt}u$$

$$\frac{d}{dt}\sqrt{L^2 - r^2 \cdot \sin(\theta)^2} = \frac{1}{2 \cdot \sqrt{L^2 - r^2 \cdot \sin(\theta)^2}} \cdot \frac{d}{dt}(L^2 - r^2 \cdot \sin(\theta)^2) = \frac{1}{2 \cdot \sqrt{L^2 - r^2 \cdot \sin(\theta)^2}} \cdot \left[\frac{d}{dt}L^2 - \frac{d}{dt}(r^2 \cdot \sin(\theta)^2) \right]$$

or
$$\frac{1}{2 \cdot \sqrt{L^2 - r^2 \cdot \sin(\theta)^2}} \cdot \left[0 - \left[\left(r^2 \right) \cdot \frac{d}{dt}\sin(\theta)^2 + \sin(\theta)^2 \cdot \frac{d}{dt}r^2 \right] \right] = \frac{1}{2 \cdot \sqrt{L^2 - r^2 \cdot \sin(\theta)^2}} \cdot \left[0 - \left[r^2 \cdot \left(\frac{d}{dt}\sin(\theta)^2 \right) + 0 \right] \right]$$

remembering:

$$\frac{d}{dt}L^2 = \frac{d}{dt}r^2 = 0$$

since L and r are
unchanging.

or
$$\frac{1}{2\sqrt{L^2 - r^2 \cdot \sin(\theta)^2}} \cdot \left[-r^2 \cdot \left[(\sin(\theta)) \cdot \frac{d}{dt} \sin(\theta) + \sin(\theta) \cdot \frac{d}{dt} \sin(\theta) \right] \right] = \frac{1}{2\sqrt{L^2 - r^2 \cdot \sin(\theta)^2}} \cdot \left[-2 \cdot r^2 \cdot \left[\sin(\theta) \cdot \left(\frac{d}{dt} \sin(\theta) \right) \right] \right]$$

or
$$\frac{1}{2\sqrt{L^2 - r^2 \cdot \sin(\theta)^2}} \cdot \left[-2 \cdot r^2 \cdot \left[\sin(\theta) \cdot \left[\left(\frac{d}{dt} \theta \right) \cdot \cos(\theta) \right] \right] \right] = \frac{1}{2\sqrt{L^2 - r^2 \cdot \sin(\theta)^2}} \cdot \left[-2 \cdot r^2 \cdot \left[\sin(\theta) \cdot \left[(\omega) \cdot \cos(\theta) \right] \right] \right]$$

or
$$\frac{-2 \cdot r^2 \cdot \omega \cdot \sin(\theta) \cdot \cos(\theta)}{2\sqrt{L^2 - r^2 \cdot \sin(\theta)^2}} = \frac{-r^2 \cdot \omega \cdot \sin(\theta) \cdot \cos(\theta)}{\sqrt{L^2 - r^2 \cdot \sin(\theta)^2}}$$

so...
$$\frac{d}{dt} h(\theta) = \text{Part}_1 + \text{Part}_2 = (-r \cdot \omega \cdot \sin(\theta)) + \frac{-r^2 \cdot \omega \cdot \sin(\theta) \cdot \cos(\theta)}{\sqrt{L^2 - r^2 \cdot \sin(\theta)^2}}$$

Example: Assume a 10,000 RPM: $\omega := \frac{10000 \cdot 360 \cdot \text{deg}}{\text{min}}$ $\omega = 1047 \frac{1}{\text{s}}$ and break the velocity equation into parts

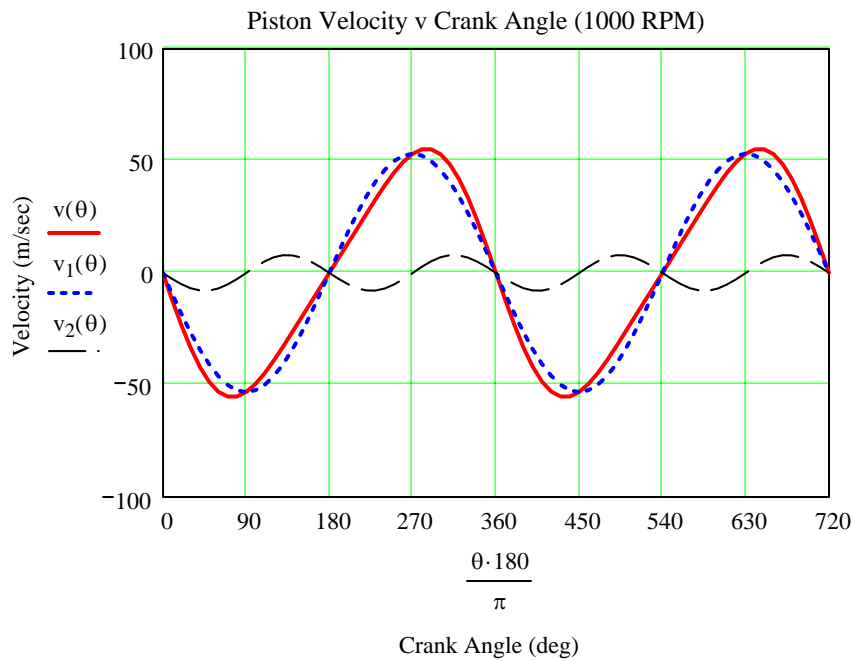
$$v_1(\theta) := -r \cdot \omega \cdot \sin(\theta)$$

$$v_2(\theta) := \frac{-r^2 \cdot \omega \cdot \sin(\theta) \cdot \cos(\theta)}{\sqrt{L^2 - r^2 \cdot \sin(\theta)^2}}$$

$$v(\theta) := (-r \cdot \omega \cdot \sin(\theta)) + \frac{-r^2 \cdot \omega \cdot \sin(\theta) \cdot \cos(\theta)}{\sqrt{L^2 - r^2 \cdot \sin(\theta)^2}} \quad \text{(the complete velocity equation)}$$

Graphing this function over the domain:

$$\theta := 0 \cdot \text{deg}, 10 \cdot \text{deg} .. 720 \cdot \text{deg}$$



Maximum velocity occurs at 74.84 degrees and 285.16 degrees after Top Dead Center as shown in the calculations (roots of the velocity curve) below:

$$v_{\max}(\theta) := \frac{d}{d\theta} v(\theta)$$

using an initial guess of:

$$\theta := 85 \cdot \text{deg} \quad \text{root}(v_{\max}(\theta), \theta) = 74.84 \text{ deg}$$

using an initial guess of:

$$\theta := 300 \cdot \text{deg} \quad \text{root}(v_{\max}(\theta), \theta) = 285.16 \text{ deg}$$

Derive the piston acceleration (α) equation. Acceleration is just the time rate of change of the velocity $v(\theta)$.

Given:
$$v(\theta) = (-r \cdot \omega \cdot \sin(\theta)) + \frac{-r^2 \cdot \omega \cdot \sin(\theta) \cdot \cos(\theta)}{\sqrt{L^2 - r^2 \cdot \sin(\theta)^2}}$$

Taking the derivative of the velocity with respect to time:

$$\frac{d}{dt}v(\theta) = \frac{d}{dt}(-r \cdot \omega \cdot \sin(\theta)) + \frac{d}{dt} \frac{-r^2 \cdot \omega \cdot \sin(\theta) \cdot \cos(\theta)}{\sqrt{L^2 - r^2 \cdot \sin(\theta)^2}}$$

remember: $\alpha = \frac{d^2}{dt^2}\theta = \frac{d}{dt}\omega$ (the time rate of change of ω (RPM) is angular acceleration - "revving the engine")

assume: $\alpha := 0 \cdot \frac{\text{rad}}{\text{sec}^2}$ (a constant RPM - no "revving" the engine)

Breaking the derivation into parts:

Part 1
$$\frac{d}{dt}(-r \cdot \omega \cdot \sin(\theta)) = \left[-r \cdot \omega \cdot \left(\frac{d}{dt} \sin(\theta) \right) \right] + -r \cdot \sin(\theta) \cdot \frac{d}{dt}\omega + \omega \cdot \sin(\theta) \cdot \frac{d}{dt}-r = (-r \cdot \omega \cdot \omega \cdot \cos(\theta)) + -r \cdot \sin(\theta) \cdot \alpha + 0 = -r \cdot \omega^2 \cdot \cos(\theta) - r \cdot \sin(\theta) \cdot \alpha$$

Letting: $p_1(\theta) := -r \cdot \omega^2 \cdot \cos(\theta) - r \cdot \sin(\theta) \cdot \alpha$

Part 2
$$\frac{d}{dt} \frac{-r^2 \cdot \omega \cdot \sin(\theta) \cdot \cos(\theta)}{\sqrt{L^2 - r^2 \cdot \sin(\theta)^2}} = \frac{d}{dt} \left[\left(-r^2 \cdot \omega \cdot \sin(\theta) \cdot \cos(\theta) \right) \cdot \left(L^2 - r^2 \cdot \sin(\theta)^2 \right)^{\frac{-1}{2}} \right]$$

Part 2.1
$$-r^2 \cdot \omega \cdot \sin(\theta) \cdot \left(L^2 - r^2 \cdot \sin(\theta)^2 \right)^{\frac{-1}{2}} \cdot \frac{d}{dt} \cos(\theta) = -r^2 \cdot \omega \cdot \sin(\theta) \cdot \left(L^2 - r^2 \cdot \sin(\theta)^2 \right)^{\frac{-1}{2}} \cdot (-\omega \cdot \sin(\theta))$$

Letting:
$$p_{21}(\theta) := -r^2 \cdot \omega \cdot \sin(\theta) \cdot \left(L^2 - r^2 \cdot \sin(\theta)^2 \right)^{\frac{-1}{2}} \cdot (-\omega \cdot \sin(\theta))$$

Part 2.2
$$-r^2 \cdot \omega \cdot \cos(\theta) \cdot \left(L^2 - r^2 \cdot \sin(\theta)^2 \right)^{\frac{-1}{2}} \cdot \left(\frac{d}{dt} \sin(\theta) \right) = -r^2 \cdot \omega \cdot \cos(\theta) \cdot \left(L^2 - r^2 \cdot \sin(\theta)^2 \right)^{\frac{-1}{2}} \cdot (\omega \cdot \cos(\theta))$$

Letting:
$$p_{22}(\theta) := -r^2 \cdot \omega \cdot \cos(\theta) \cdot \left(L^2 - r^2 \cdot \sin(\theta)^2 \right)^{\frac{-1}{2}} \cdot (\omega \cdot \cos(\theta))$$

Part 2.3
$$-r^2 \cdot \left(L^2 - r^2 \cdot \sin(\theta)^2 \right)^{\frac{-1}{2}} \cdot \sin(\theta) \cdot \cos(\theta) \cdot \frac{d}{dt} \omega = -r^2 \cdot \left(L^2 - r^2 \cdot \sin(\theta)^2 \right)^{\frac{-1}{2}} \cdot \sin(\theta) \cdot \cos(\theta) \cdot \alpha$$

Letting:
$$p_{23}(\theta) := -r^2 \cdot \left(L^2 - r^2 \cdot \sin(\theta)^2 \right)^{\frac{-1}{2}} \cdot \sin(\theta) \cdot \cos(\theta) \cdot \alpha$$

Part 2.4 $(\omega \cdot \sin(\theta) \cdot \cos(\theta)) \cdot (L^2 - r^2 \cdot \sin(\theta)^2)^{\frac{-1}{2}} \cdot \frac{d}{dt} - r = 0$ **(since crank throw does not change with time)**

Letting: $p_{24}(\theta) := 0 \cdot \frac{ft}{\text{sec}^2}$

Part 2.5 $-r^2 \cdot \omega \cdot \sin(\theta) \cdot \cos(\theta) \cdot \frac{d}{dt} (L^2 - r^2 \cdot \sin(\theta)^2)^{\frac{-1}{2}} = -r^2 \cdot \omega \cdot \sin(\theta) \cdot \cos(\theta) \cdot \frac{-1}{2} \cdot (L^2 - r^2 \cdot \sin(\theta)^2)^{\frac{-3}{2}} \cdot \frac{d}{dt} (L^2 - r^2 \cdot \sin(\theta)^2)$

$$-r^2 \cdot \omega \cdot \sin(\theta) \cdot \cos(\theta) \cdot \frac{d}{dt} (L^2 - r^2 \cdot \sin(\theta)^2)^{\frac{-1}{2}} = -r^2 \cdot \omega \cdot \sin(\theta) \cdot \cos(\theta) \cdot \frac{-1}{2} \cdot (L^2 - r^2 \cdot \sin(\theta)^2)^{\frac{-3}{2}} \cdot [0 + (-2) \cdot (r^2) \cdot [\sin(\theta) \cdot [(\omega) \cdot \cos(\theta)]]]$$

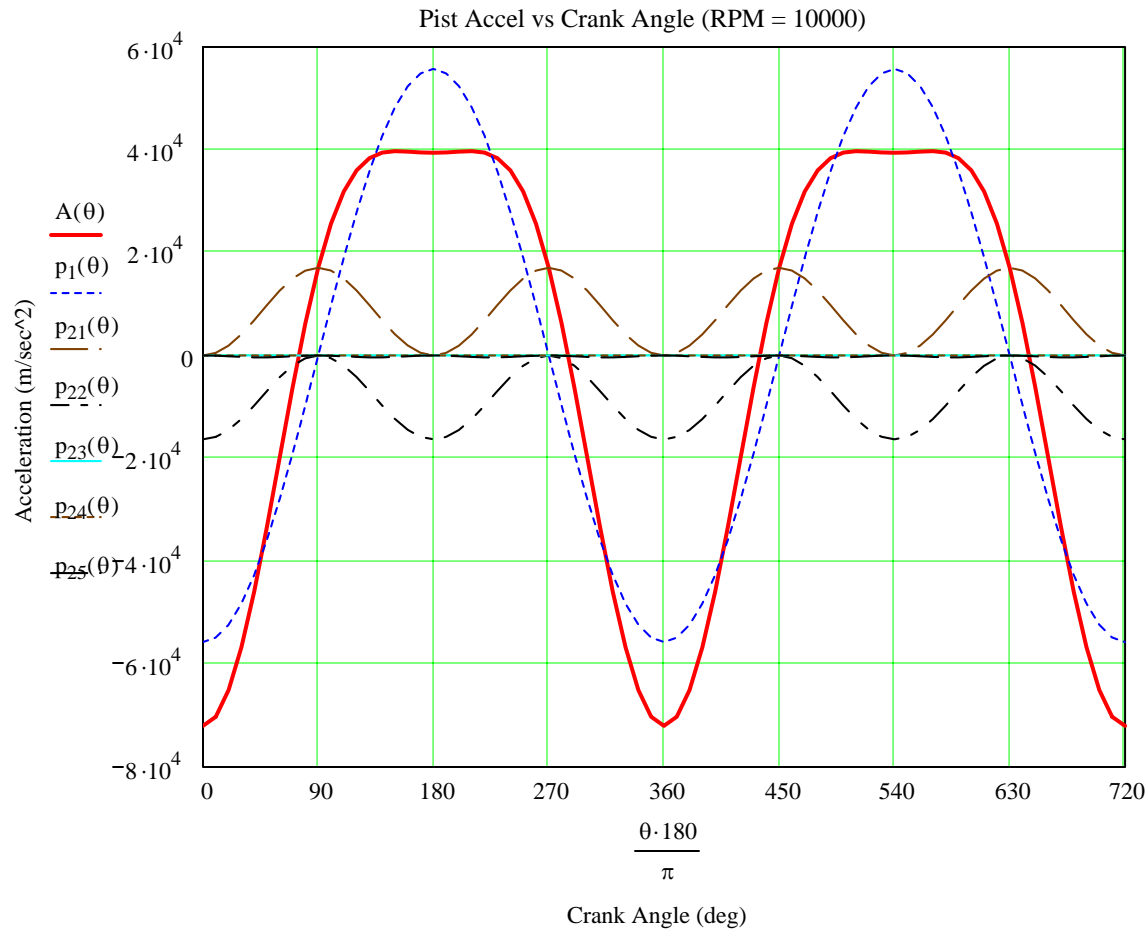
Letting: $p_{25}(\theta) := -r^2 \cdot \omega \cdot \sin(\theta) \cdot \cos(\theta) \cdot \frac{-1}{2} \cdot (L^2 - r^2 \cdot \sin(\theta)^2)^{\frac{-3}{2}} \cdot [0 + (-2) \cdot (r^2) \cdot [\sin(\theta) \cdot [(\omega) \cdot \cos(\theta)]]]$

Simplifying: $p_{25}(\theta) := r^4 \cdot \omega^2 \cdot (-1 + \cos(\theta)^2) \cdot \frac{\cos(\theta)^2}{(L^2 - r^2 + r^2 \cdot \cos(\theta)^2)^{\frac{3}{2}}}$

Adding the parts together to get the complete acceleration function and graphing it over the domain:

$\theta := 0 \cdot \text{deg}, 10 \cdot \text{deg} \dots 720 \cdot \text{deg}$

$$A(\theta) := p_1(\theta) + p_{21}(\theta) + p_{22}(\theta) + p_{23}(\theta) + p_{24}(\theta) + p_{25}(\theta)$$



example acceleration at 220 degrees past TDC:

$$A(220 \cdot \text{deg}) = 39436 \frac{\text{m}}{\text{s}^2}$$

example g calculation:

$$\frac{A(220 \cdot \text{deg})}{g} = 4021$$

Connecting rod stress analysis:

Define: $kPa := 1000 \cdot Pa$

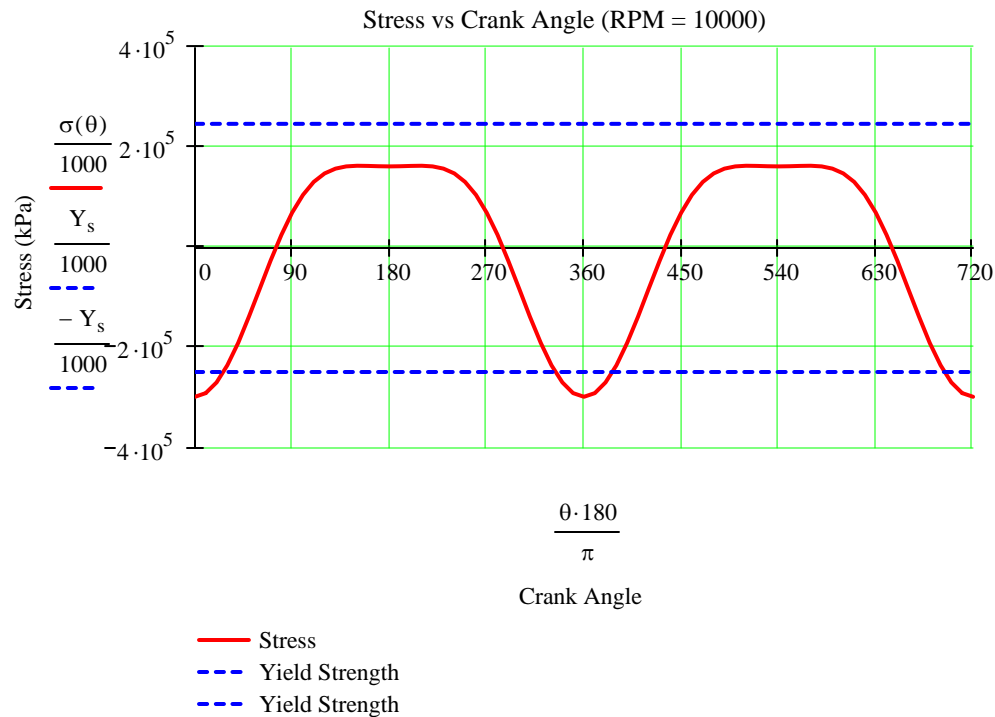
graphing domain:

$\theta := 0 \cdot deg, 10 \cdot deg .. 720 \cdot deg$

Yield Strength of rod steel: $Y_s := 36000 \cdot psi$ $Y_s = 248211 \text{ kPa}$

Variables: $cross_{sec} := 0.51 \cdot in^2$ $Mass_{piston} := 3 \cdot lb$ $\omega = 1047 \frac{1}{s}$ **10,000 RPM**

Using Newton's 2nd Law: $F = ma$ $F(\theta) := Mass_{piston} \cdot A(\theta)$ $\sigma(\theta) := \frac{F(\theta)}{cross_{sec}}$



Finding failure crank angles using Mathcad's root finding function with initial guesses of:

$\theta := -10 \cdot deg$ $root(\sigma(\theta) + Y_s, \theta) = -26 \cdot deg$

$\theta := 10 \cdot deg$ $root(\sigma(\theta) + Y_s, \theta) = 26 \cdot deg$

Conclusion: For the given geometry and parameters (RPM = 10,000), the failure will occur near TDC (from 26 degrees before until 26 degrees after TDC)

Graphing in 3D and contour plot the piston acceleration (α) equation as a function of crank angle (θ) and engine RPM (ω). This will be a 3-D analysis!

Remembering:
$$v(\theta, \omega) = (-r \cdot \omega \cdot \sin(\theta)) + \frac{-r^2 \cdot \omega \cdot \sin(\theta) \cdot \cos(\theta)}{\sqrt{L^2 - r^2 \cdot \sin(\theta)^2}}$$

Taking the derivative of the velocity with respect to time:

$$\frac{d}{dt}v(\theta, \omega) = \frac{d}{dt}(-r \cdot \omega \cdot \sin(\theta)) + \frac{d}{dt} \frac{-r^2 \cdot \omega \cdot \sin(\theta) \cdot \cos(\theta)}{\sqrt{L^2 - r^2 \cdot \sin(\theta)^2}}$$

remember:
$$\alpha = \frac{d^2}{dt^2}\theta = \frac{d}{dt}\omega$$

assume:
$$\alpha := 0 \cdot \frac{\text{rad}}{\text{sec}^2} \quad (\text{a constant RPM})$$

Breaking the derivation into parts:

Part 1
$$\frac{d}{dt}(-r \cdot \omega \cdot \sin(\theta)) = \left[-r \cdot \omega \cdot \left(\frac{d}{dt} \sin(\theta) \right) \right] + -r \cdot \sin(\theta) \cdot \frac{d}{dt}\omega + \omega \cdot \sin(\theta) \cdot \frac{d}{dt}-r = (-r \cdot \omega \cdot \omega \cdot \cos(\theta)) + -r \cdot \sin(\theta) \cdot \alpha + 0 = -r \cdot \omega^2 \cdot \cos(\theta) - r \cdot \sin(\theta) \cdot \alpha$$

Letting:
$$p_1(\theta, \omega) := -r \cdot \omega^2 \cdot \cos(\theta) - r \cdot \sin(\theta) \cdot \alpha$$

Part 2
$$\frac{d}{dt} \frac{-r^2 \cdot \omega \cdot \sin(\theta) \cdot \cos(\theta)}{\sqrt{L^2 - r^2 \cdot \sin(\theta)^2}} = \frac{d}{dt} \left[\frac{-1}{\left(-r^2 \cdot \omega \cdot \sin(\theta) \cdot \cos(\theta) \right) \cdot \left(L^2 - r^2 \cdot \sin(\theta)^2 \right)^{\frac{1}{2}}} \right]$$

Part 2.1
$$-r^2 \cdot \omega \cdot \sin(\theta) \cdot \left(L^2 - r^2 \cdot \sin(\theta)^2 \right)^{\frac{-1}{2}} \cdot \frac{d}{dt} \cos(\theta) = -r^2 \cdot \omega \cdot \sin(\theta) \cdot \left(L^2 - r^2 \cdot \sin(\theta)^2 \right)^{\frac{-1}{2}} \cdot (-\omega \cdot \sin(\theta))$$

Letting:
$$p_{21}(\theta, \omega) := -r^2 \cdot \omega \cdot \sin(\theta) \cdot \left(L^2 - r^2 \cdot \sin(\theta)^2 \right)^{\frac{-1}{2}} \cdot (-\omega \cdot \sin(\theta))$$

Part 2.2
$$-r^2 \cdot \omega \cdot \cos(\theta) \cdot \left(L^2 - r^2 \cdot \sin(\theta)^2 \right)^{\frac{-1}{2}} \cdot \left(\frac{d}{dt} \sin(\theta) \right) = -r^2 \cdot \omega \cdot \cos(\theta) \cdot \left(L^2 - r^2 \cdot \sin(\theta)^2 \right)^{\frac{-1}{2}} \cdot (\omega \cdot \cos(\theta))$$

Letting:
$$p_{22}(\theta, \omega) := -r^2 \cdot \omega \cdot \cos(\theta) \cdot \left(L^2 - r^2 \cdot \sin(\theta)^2 \right)^{\frac{-1}{2}} \cdot (\omega \cdot \cos(\theta))$$

Part 2.3
$$\left[-r^2 \cdot \left(L^2 - r^2 \cdot \sin(\theta)^2 \right)^{\frac{-1}{2}} \cdot \sin(\theta) \cdot \cos(\theta) \cdot \frac{d}{dt} \omega \right] = -r^2 \cdot \left(L^2 - r^2 \cdot \sin(\theta)^2 \right)^{\frac{-1}{2}} \cdot \sin(\theta) \cdot \cos(\theta) \cdot \alpha$$

Letting:
$$p_{23}(\theta, \omega) := -r^2 \cdot (L^2 - r^2 \cdot \sin(\theta)^2)^{\frac{-1}{2}} \cdot \sin(\theta) \cdot \cos(\theta) \cdot \alpha$$

Part 2.4
$$(\omega \cdot \sin(\theta) \cdot \cos(\theta)) \cdot (L^2 - r^2 \cdot \sin(\theta)^2)^{\frac{-1}{2}} \cdot \frac{d}{dt} - r = 0 \quad \text{(since crank throw does not change with time)}$$

so... letting:
$$p_{24}(\theta, \omega) := 0 \cdot \frac{ft}{sec^2}$$

Part 2.5
$$\left[-r^2 \cdot \omega \cdot \sin(\theta) \cdot \cos(\theta) \cdot \frac{d}{dt} (L^2 - r^2 \cdot \sin(\theta)^2)^{\frac{-1}{2}} \right] = -r^2 \cdot \omega \cdot \sin(\theta) \cdot \cos(\theta) \cdot \frac{-1}{2} \cdot (L^2 - r^2 \cdot \sin(\theta)^2)^{\frac{-3}{2}} \cdot \frac{d}{dt} (L^2 - r^2 \cdot \sin(\theta)^2)$$

$$\left[-r^2 \cdot \omega \cdot \sin(\theta) \cdot \cos(\theta) \cdot \frac{d}{dt} (L^2 - r^2 \cdot \sin(\theta)^2)^{\frac{-1}{2}} \right] = -r^2 \cdot \omega \cdot \sin(\theta) \cdot \cos(\theta) \cdot \frac{-1}{2} \cdot (L^2 - r^2 \cdot \sin(\theta)^2)^{\frac{-3}{2}} \cdot [0 + (-2) \cdot (r^2) \cdot [\sin(\theta) \cdot [(\omega) \cdot \cos(\theta)]]]$$

Letting:
$$p_{25}(\theta) := -r^2 \cdot \omega \cdot \sin(\theta) \cdot \cos(\theta) \cdot \frac{-1}{2} \cdot (L^2 - r^2 \cdot \sin(\theta)^2)^{\frac{-3}{2}} \cdot [0 + (-2) \cdot (r^2) \cdot [\sin(\theta) \cdot [(\omega) \cdot \cos(\theta)]]]$$

Simplifying:
$$p_{25}(\theta, \omega) := \left[-r^4 \cdot \omega^2 \cdot (-1 + \cos(\theta)^2) \cdot \frac{\cos(\theta)^2}{(L^2 - r^2 + r^2 \cdot \cos(\theta)^2)^{\frac{3}{2}}} \right]$$

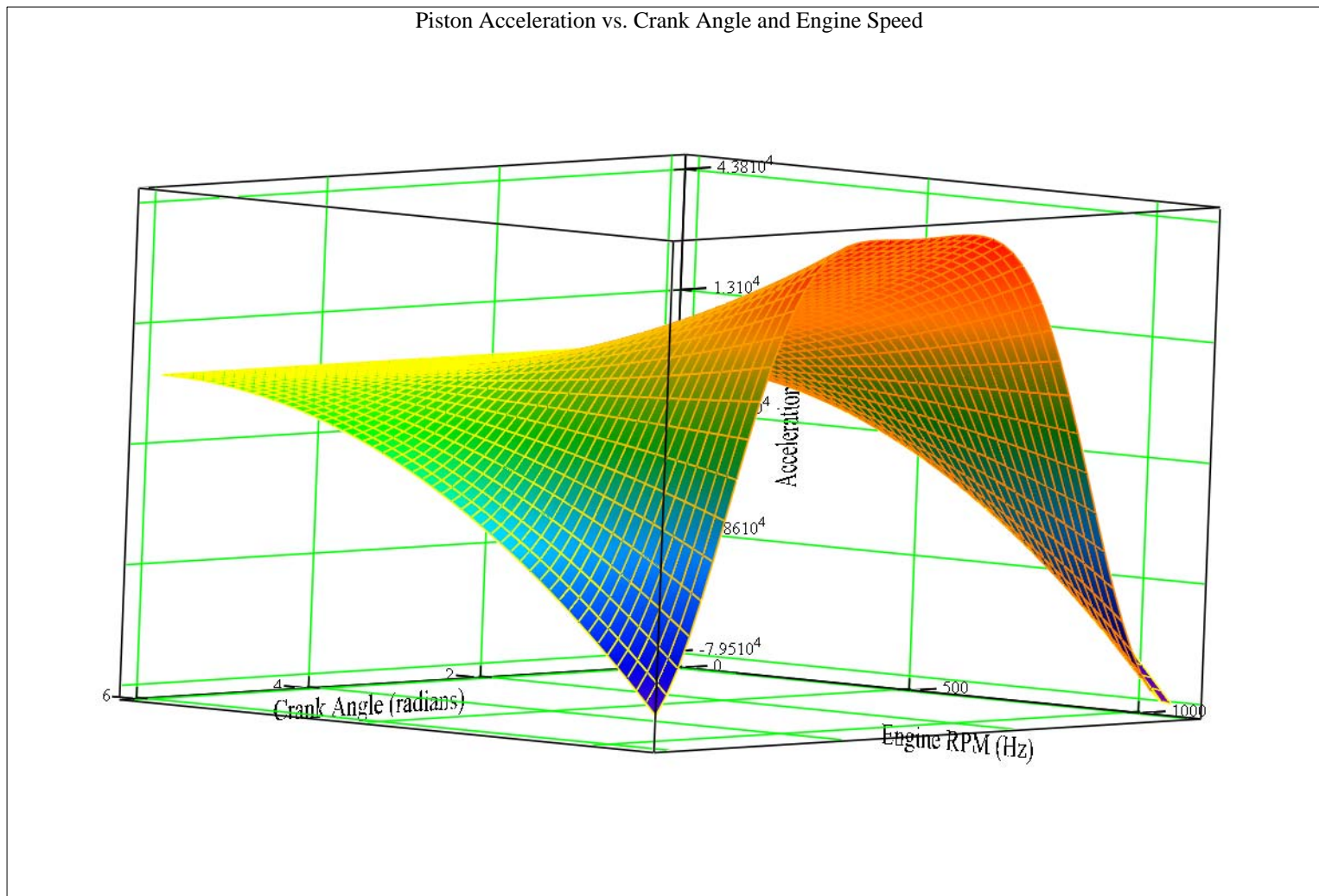
Adding the parts together to get the single (and complete) acceleration function as a function of crank angle (θ) and engine RPM (ω) [and graphing it over the domain shown to the right] yields:

$$0 \leq \theta \leq 6 \quad (\text{radians})$$

$$0 \leq \omega \leq 1100 \quad (\text{Hz}) \quad \text{where: } 1100 \text{ Hz} = 10504 \text{ RPM}$$

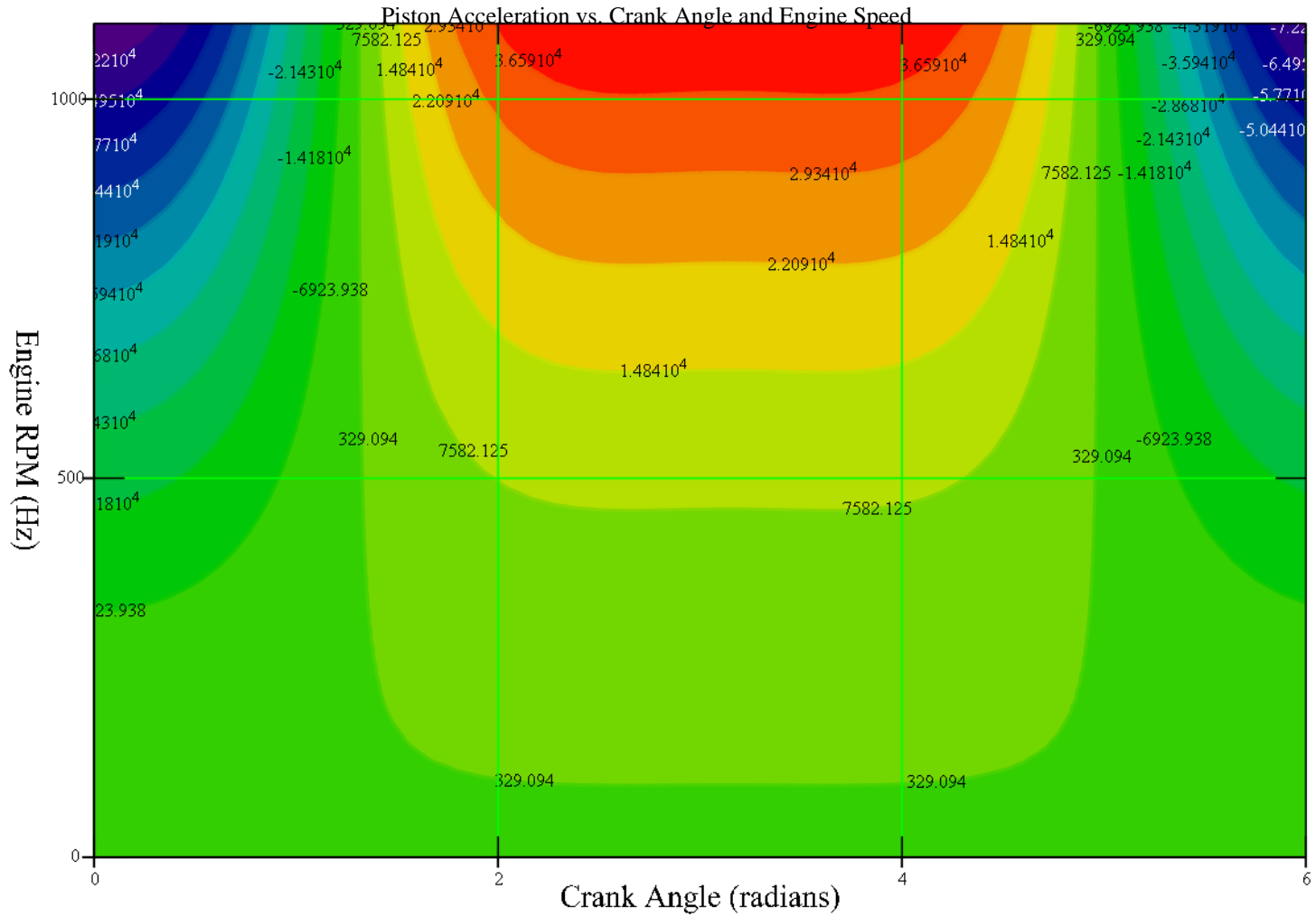
$$A(\theta, \omega) := p_1(\theta, \omega) + p_{21}(\theta, \omega) + p_{22}(\theta, \omega) + p_{23}(\theta, \omega) + p_{24}(\theta, \omega) + p_{25}(\theta, \omega)$$

3-D Plot



A

**Contour Plot
of piston
acceleration
as a function
of crank
angle and
engine
speed (RPM)**



A